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# Finite Element Analysis of Piston Using Thermal Barrier Coating

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# **ABSTRACT**

The piston is responsible for transmitting gas pressure exerted by combustion gases into reciprocating motion and is responsible for kinetic energy. During this process, the piston crown is continuously subjected to various thermal and structural loads such as gas pressure load, side thrust, and connecting rod load, inertia load. This will result in piston seizing by overheating, and cracks on top land ultimately affecting engine efficiency. To reduce heat absorption and increase insulating property Thermal barrier coating is applied on top land. The modeling of the piston is made on Solid works. In this project, the Analysis is performed on a piston made of aluminum alloy with and without thermal barrier coating materials (Magnesium zirconate, Lanthanum doped zirconate, Yttria stabilized zirconia). Structural analysis, Steady-state thermal analysis is carried out on the piston in ANSYS and aims to study its behavior by comparing it with the conventional piston.

Keywords: Thermal Barrier Coating, Structural Analysis, Steady State Thermal Analysis

#### 1. INTRODUCTION

In an internal combustion engine piston is the reciprocating part that transmits force generated by combustion of air fuel mixture. It occurs by pushing the piston through great amount force which is produced by expansion of combustion gases. In a conventional IC engine air -fuel mixture of proportioned quantity enters in the cylinder when piston moving towards bottom dead center and intake takes place. At the time of compression stage air-fuel mixture is compressed to high limits of temperature and pressure, at the end of stroke charge ignites and combustion of charge results in expansion of combustion gases which pushes the piston down at great speed and piston transmits this energy to piston pin, connecting rod, crank shaft. Due to this impact various thermal forces, Inertia force, combustion

gas pressure, connecting rod load and side thrust, acts on piston, and most of them acts on top land or crown of piston.

Throughout this process most of the heat produced is absorbed by the piston. To prevent piston from failing, cracks on crown, piston seizure due to overheating, a thermal barrier coating on piston surface is utilized. Thermal barrier coating acts as insulator which acts as barrier for heat to transmit into piston. The materials used for coating have low thermal conducting property and high coefficient of thermal expansion, Retain its structure and phase at elevated temperature. The study indicates that 30% of heat is carried away by engine cooling system and good solution may be engine thermal barrier coating. Thermal barrier coating owing their insulating properties which helps in increasing the fuel

economy as fuel burns more due increased temperature in the region, which in turn increase engine efficiency. Thermal barrier coating reduces hydrocarbon, reduces emission by producing less smoke and carbon monoxides. Another study indicates that significant increase in temperature results in increase in the thermal efficiency of engine. This will reduce the heat loss from piston as a result of burning un-burnt gases.

Thermal barrier coating reportedly modifies prior condition for combustion of fuel will result in lessen ignition delay time and shortening fuel consumption.

### 2. REALATED WORK

OBJECTIVE: The objective of this project is to increase the insulating property of piston by coating various types of TBC material on piston crown namely La1.7Dy0.3Zr2O7, MgZrO3 and Yttria stabilized zirconia (YSZ) and perform comparative study by varying thickness of coating. By performing static structural analysis and Steady state thermal analysis in ANSYS most effective thermal barrier coating material is identified.

### PROBLEM IDENTIFICATION:

- In the internal combustion engine, piston subjected to fatigue load, gas pressure load and side thrust.
- · Engine cooling system carries away 30 percentage of heat which is direct loss.
- ·Overheating of piston result ultimately piston seizure ·Un-burnt fuel mixture cause reduction in engine efficiency

### LITERATURE REVIEW:

[1] Thermal barrier coating on IC Engine Piston to enhance better utilization of heat produced inside the combustion chamber ultimately improving engine's efficiency

Year of publication: 2017

Author: Mohsin Attar, Prof.Mr. Ajay Bhongade

Observation: This research is aimed to improve efficiency of engine with utilizing Yttria stabilized zirconia as thermal barrier coating material. This research considered 150 cc engine piston and compared with yttria stabilized zirconia coating with various thickness on top land of piston. This research suggested that using YSZ coating of 0.6mm reduces heat flux and temperature on crown of piston.

[2] Analysis on a thermal barrier coated (TBC) piston in a single cylinder diesel engine powered by Jatropha biodiesel–diesel blends

Author: Harish Venu, Prabhu Appavu

Year of publication: 2019

Observation: This research work aimed to improve diesel efficiency operated with Jatropha bio diesel blends using Yttria stabilized zirconia coating (YSZ). Yttria stabilized zirconia of 200 micro meter thick layer is coated by plasma spray technique at temperature 8300°C on piston operated on JB20(20% jatropha biodiesel and 80% diesel fuel). This research indicates in improving combustion characteristics and lowering Hydro carbon, carbon dioxide emissions.

[3] Design and static thermal analysis of piston using thermal barrier coating materials using fem

Author: Mr. Vatti Sairam Krishna, Mr.K. Muralidhar

Year of publication: 2021

Observation:

In this research thermal analysis is conducted on piston of 150cc engine of materials Al402, Ti64AlV coated with Thermal barrier coating materials, Mgzro3, Ni-Cr-Al of 0.4mm and suggested that Al-Sic piston coated Ni-Cr-Al Is efficient and suited for high temperature application.

[4]Design and Thermal Analysis of Ceramic Coated Diesel Engine Piston (Mgzro3 & Nicral)

Author:M.Mathanbabu,P.Mohanraj,S.Navaneetha,Krish nan ,T.Naveen Kumar ,S.Vijay

Year of publication: 2019

Observation: In this research Mahindra Scorpio Group M11B15-Piston is considered and piston material as Alsic and steel is assumed. Thermal analysis is conducted for uncoated piston and piston coated with thermal barrier coating materials MgZrO3 and NiCrAl as bond coat. This research compared results of analysis and suggested that piston coated with MgZrO3 has better insulating property.

[5] Static and Thermal Analysis of Aluminium Alloy Piston with Ceramic Coating

Author: G.Venkata Srinivasa Rao, CH.Venkateswara Rao, T.N.Charyulu, Sk.Nakki Jani, N.Ch.Sai Prathap

Year of publication:2019

Observation: In this research paper piston is designed in catia v5 and the model is then imported to ansys and in ansys geometric model is converted into FEA model and apply boundary conditions to find the stress, temperature distribution and heat flux and piston is

coated with two thermal barrier coatings those are ALSIC(aluminium silicon) and NICRAL(nickel chromium aluminium). They concluded that based on their analysis aluminium silicon carbide has more temperature resistance and less amount of heat flux as compared with aluminium alloys.

[6] Design and thermal analysis of ceramic coated aluminium alloy piston

Author: T. G. Thiyagarajan, A. Velladurai

Year of publication: 2017

Observation: In this paper they mainly focus on determining the temperature distribution of a ceramic coated piston. In this research they used various materials to coat piston like mullite, yttrium stabilized zirconia(Y-PSZ), and Magnesium Stabilized Zirconia (Mg-PSZ) are used as ceramic coatings. Based on the analysis the uncoated piston has less performance has compared with the coated one. And they found that magnesium stabilized zirconia has similar properties of Yttrium stabilized zirconia which is best suitable for TBC material.

[7] Investigation on effect of Yttria Stabilized Zirconia coated piston crown on performance and emission characteristics of a diesel engine

Author: G. Sivakumar, S. Senthil Kumar

Year of publication: 2014

Observation: In this paper experimental investigation made on three-cylinder diesel engine with piston coated with Yittria stabilized zirconia and performance and emissive characteristics are studied. Based on study TBC material coated engine has improved Brake specific fuel consumption and hydrocarbon emission reduced up to 35.27% and carbon monoxide reduction up to 2.7%.

[8] Thermo-Mechanical Analysis of a Piston with Thermal Barrier Coatings (Tbc)

Author: Abhinav Lal, Mr. Morrish kumar

Year of publication: 2017

Observation: In this paper FEA analysis and thermal analysis is conducted on piston of Al-sic and Titanium material and coated with MgZrO3 and Yittria partially stabilized zirconia. This research suggest that Al-sic coated with MgZrO3 of 0.5 mm thickness retained its Temperature near to titanium piston.

[9]Thermal Behaviorand Optimization of Piston Coating

Material (Al-Si) Used in Petrol Engine

Author: M. Sivanesan and C. Vinothkumar

Year of publication:2016

Observation: In this paper piston is with various amount of thickness from 0.2 to 1.6mm without considering bond carbidecoating layers. In this research paper they are mainly focusing on thermal stress distribution of piston at real engine during combustion. They conduct thermal and structural analysis by coating with different materials like aluminium oxide, zirconium oxide and mullite. Finally, they concluded that aluminium oxide has higher efficiency as compared with other materials

### 3. PROPOSED WORK

# DESIGN:

Thickness of piston head	10.378 mm
Radial thickness of piston	2.59 mm
ring	
Axial thickness of piston	2.08 mm
ring	
No of piston rings	4
Width of top land	11.415 mm
Width of ring land	1.664 mm
Radial depth of piston	2.99 mm
ring groove	S S
Th <mark>ickn</mark> ess of pi <mark>ston barr</mark> el	10.0 <mark>1 m</mark> m
at <mark>top e</mark> nd	
Pi <mark>ston p</mark> in diameter	25.2 mm
Diameter of piston boss	43.434 mm
Length of Skirt	50.4 mm
Total length of piston	75.126 mm
Thickness of piston barrel	2.505 mm
at open end	

In this work we have considered royal Enfield 500cc engine which dimensions as bore B=84mm and stroke l=90mm, maximum power=27.184 bhp at 5250 rpm,maximum torque=41.3 Nm at 4000 rpm

DIMENSIONS OF ROYAL ENFIELD 500cc Engine Piston

Table: 1

MODELLING IN SOLID WORKS: In this project we have considered Royal Enfield 500cc engine piston of material Aluminium alloy and designed using Solid Works. Ceramic material of 0.3mm and 0.5mm thick layers are modeled.

Piston without Thermal barrier coating:

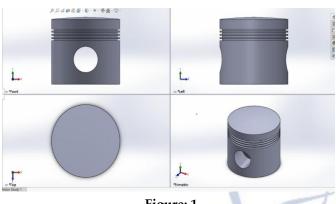


Figure: 1

Piston with Thermal barrier coating:

Figure:2

# MATERIAL INFORMATION

Based on the properties and studying the literature review we are choosing three materials those are Magnesium zirconate (MgZrO3),lanthanum doped zirconate(La<sub>1.7</sub>Dy<sub>0.3</sub>Zr<sub>2</sub>O<sub>7</sub>, Yttria Stabilized Zirconia(YSZ) and their compositions are tabulated below.

1.Aluminium alloy(Al6061)

Derived from	Young's modulus and Poisson's ratio
Young's modulus	70000MPa
Poisson's ratio	0.33
Bulk modulus	68627MPa
Shear modulus	26316MPa
Isotropic secant coefficient	23.6 1/°C
of thermal expansion	
Tensile ultimate strength	310MPa
Tensile yield strength	276MPa
Thermal	
Isotropic thermal	0.167 W/mm°C
conductivity	
Specific heat constant	896mJ/kg°C
pressure	PU
Density	2.7× 10 <sup>-6</sup> kg/mm <sup>3</sup>

Table: 2 2. Magnesium zirconate (MgZrO3)

2. Magnesiam zirconate (MgZrO3)				
structural				
Derived from	Young's modulus and			
	Poisson's ratio			
Young's modulus	46000MPa			
Poisson's ratio	0.2			
Bulk modulus	25556MPa			

Shear modulus	19167MPa
Isotropic secant coefficient of	$2.7 \times 10^{-05}$ 1 /°C
thermal expansion	
Tensile ultimate strength	180MPa
Tensile yield strength	51MPa
Thermal	
Isotropic thermal conductivity	0.0008 W/mm°C
Specific heat constant pressure	$1.04 \times 10^6 \text{mJ/kg}^{\circ}\text{C}$
Density	$5.6 \times 10^{-6} \text{kg/mm}^3$

Table: 3

3.Lanthanum doped zirconate (La<sub>1.7</sub>Dy<sub>0.3</sub>Zr<sub>2</sub>O<sub>7</sub>)

structural			
Derived from	Young's modulus and		
- 46 4	Poisson's ratio		
Young's modulus	$1.75 \times 10^{5} MPa$		
Poisson's ratio	0.28		
Bulk modulus	$1.3258 \times 10^{5} MPa$		
Shear modulus	68359MPa		
Isotropic secant	$1.035 \times 10^{-05}$ 1/°C		
coefficient of thermal			
expansion			
Thermal			
Isotropic thermal	0.000468 W/mm°C		
conductivity			
Specific heat constant	$1.356 \times 10^6 \text{mJ/kg}^{\circ}\text{C}$		
pressure			
<b>Density</b>	$5.6 \times 10^{-6} \text{kg/mm}^3$		

# Table: 4

# 4.Yttria Stabilized Zirconia (YSZ)

structural					
Derived from	Young's modulus and				
	Poisson's ratio				
Young's modulus	$2.1 \times 10^{5} MPa$				
Poisson's ratio	0.3				
Bulk modulus	$1.75 \times 10^{5} MPa$				
Shear modulus	80769MPa				
Isotropic secant	$6.69 \times 10^{-05}$ 1/°C				
coefficient of thermal					
expansion					
Tensile ultimate strength	690MPa				
Tensile yield strength	550MPa				
Thermal	97				
Isotropic thermal	0.0022 W/mm°C				
conductivity					
Density	$6.02\times-06 \text{ kg/mm}^3$				

# Table: 5

ANALYSIS: The designed piston model from Solid works is imported to ANSYS work bench and steady state thermal analysis, static structural analysis carried out as, designed piston is imported in ansys and the materials are assigned for the piston and also for the coating on the piston crown. Meshing is done for the piston and tetrahedron elements are formed in meshing. Loads applied on the piston crown such as temperature and co efficient of convection film. Then the solver is made to run so that the solution can be obtained. Total

heat flux and temperature is obtained. Static structural analysis carried out as, Upto meshing same procedure is At static condition the loads were applied. Pressure at piston crown and the surfaces are considered to be non frictional contact. Analysis is carried to observe total deformation, stress and strain.

### **BOUNDARY CONDITIONS FOR PISTON:**

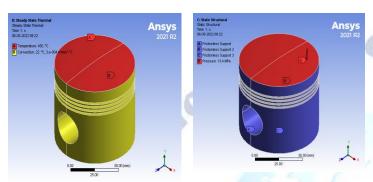


Fig: 3(a) Fig: 3(b) STEADY STATE THERMAL ANALYSIS:

Material: Al6061 + without coating

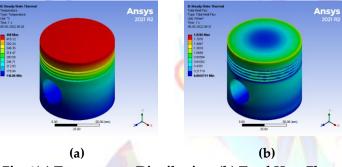


Fig: 4(a) Temperature Distribution, (b) Total Heat Flux Material: Al6061 + with MgZrO3 of 0.5 mm thickness coating

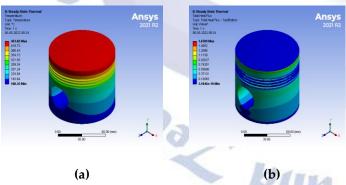
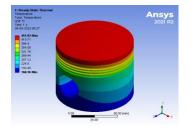
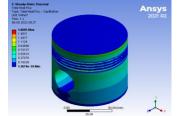
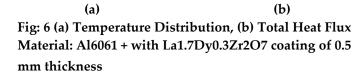


Fig: 5 (a) Temperature Distribution, (b) Total Heat Flux Material: Al6061 + with MgZrO3 of 0.3 mm thickness coating







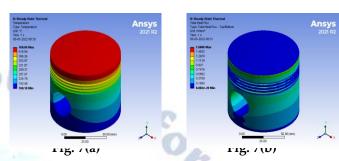


Fig: (a) Temperature Distribution, (b) Total Heat Flux Material: Al6061 + with La1.7Dy0.3Zr2O7 coating of 0.3mm thickness

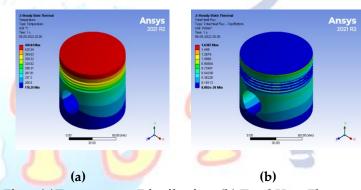


Fig: 8 (a)Temperature Distribution, (b) Total Heat Flux Material: Al6061 + with YSZ coating of 0.5 mm thickness

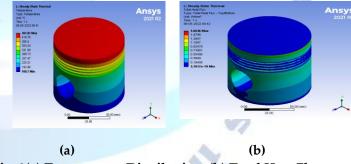


Fig: 9(a) Temperature Distribution, (b) Total Heat Flux Material: Al6061 + with YSZ coating of 0.3 mm thickness

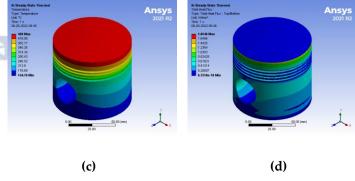


Fig: 10(c) Temperature Distribution, (d) Total Heat Flux

#### STATIC STRUCTURAL ANALYSIS:

Material: Al6061 + without coating

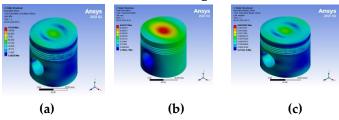


Fig: 11 (A) Equivalent Stress (B) Total Deformation (C) **Equivalent Elastic Strain** 

Material: Al6061 + with MgZrO3 of 0.5 mm thickness coating

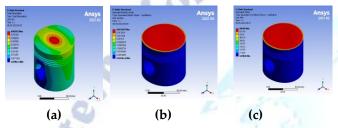


Fig: 12(a) Total Deformation (b) Equivalent Elastic Strain (c) Equivalent Stress

Material: Al6061 + with MgZrO3 of 0.3 mm thickness coating

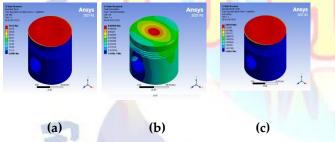


Fig: 13 (a) Equivalent Stress (b) Total Deformation (c) **Equivalent Elastic Strain** 

Material: Al6061 + with La1.7Dy0.3Zr2O7 coating of 0.5 mm thickness:

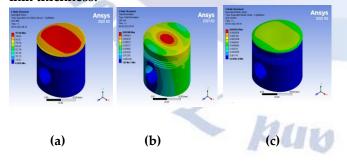


Fig: 14 (a) Equivalent Stress (b) Total Deformation (c) **Equivalent Elastic Strain** 

Material: Al6061 + with La1.7Dy0.3Zr2O7 coating of 0.3 mm thickness:

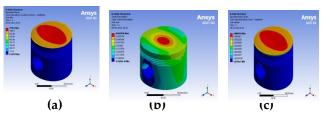


Fig: 15 (a) Equivalent Stress (b) Total Deformation (c) **Equivalent Elastic Strain** 

Material: Al6061 + with YSZ coating of 0.3 mm thickness

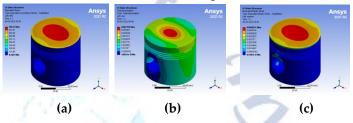


Fig: 16(a) Equivalent Stress (b) Total Deformation (c) **Equivalent Elastic Strain** 

Material: Al6061 + with YSZ coating of 0.5 mm thickness

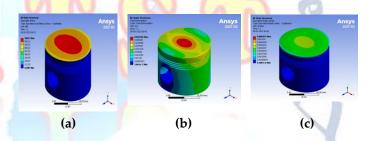


Fig: 17(a) Equivalent Stress (b) Total Deformation (c) **Equivalent Elastic Strain** 

## 4. RESULTS

From the analysis of 0.3 mm and 0.5 mm thickness of Thermal barrier coating, results of Total heat flux, Temperature distribution and von-Misses stress and equivalent strain are obtained and values are listed below Piston with 0.5 mm coating thickness

1 13ton with 0.5 mm counting thickness						
Material	Temperature°C		Heat flux W/mm <sup>2</sup>			
Al6061+TBC	Max	Min	Max	Min		
Uncoated	450	145	1.920	0.00427		
Mgzro3	450	160.5	1.6709	3.1645e-19		
Ldz	450	160.18	1.6686	8.002e-20		
Ysz	450	160.7	1.6646	3.1677e-19		

Table: 6

Materi	Total		Equivalent		Von-Mise		
al	deformation		elastic strain		s	stress	
	mm					MPa	
Al6061	max	min	max	min	ma	min	
+TBC					x		
Uncoat	0.072	3.19	0.00170	9.213	119	0.26	
ed		5e-7	5	3e-6	.28	528	
MgZrO	0.064353	3.01	0.01520	6.551	699	0.29	
3		96e-	7	e-6		09	
		8		A.			
LDZ	0.055378	2.87	0.01053	6.519	117	0.21	
		e-7	2	e-6	4.9	95	
YSZ	0.056296	2.66	0.00810	6.498	100	0.29	
	-	1e-7	15	e-19	3.6	07	

# Table: 7

# Piston with 0.3 mm coating thickness:

	A	U		
Material	Temperature°C		Heat flux W/mm <sup>2</sup>	
Al6061+TBC	Max	Max Min		Min
Mgzro3	450	160.15	1.6685	1.367e-19
Ldz	450	176.29	1.6302	8.002e-20
Ysz	450	144.78	1.8546	6.571e-18

# Table: 8

Material	Total		E <mark>quivalen</mark> t		Von-Mises	
-11	deformation		elastic strain		stress MPa	
	mm					
Al6061+	Max	Min	Max	Min	Ma	Min
TBC					x	When the
Mgzro3	0.068	7.547e	0.0152	6.594	701.	0.299
1	105	-7	54	e-6	27	89
LDZ 💮	0.062	4.320e	0.0068	6.522	119	0.292
	02	-8	516	e-6	8.2	79
YSZ	0.062	4.9281	0.0049	6.510	103	0.292
	416	e-8	215	e-6	3.2	7

Table: 9

# 5. CONCLUSION

Hence the results from analysis of Aluminum alloy piston coated with 0.5mm thickness of Yttria stabilized zirconia and Lanthanum doped zirconate has less heat flux compared with magnesium zirconate and heat flux decreases with increase in thickness of Thermal barrier coating. With decrease in heat flux results better utilization of heat therefore engine efficiency increases. From observing the Analysis results piston coated with TBC material is more efficient than uncoated piston.

#### Conflict of interest statement

Authors declare that they do not have any conflict of interest.

#### REFERENCES

- [1] S.K. Mohammad Shareef, M Sai Vikas, A.L.N Arun Kumar, Abhishek Dasore, Sanjay Chhalotre, UpendraRajak, Trikendra Nath Verma. "Design and thermal analysis of engine cylinder fin body using various fin profiles" 2214-7853/2021 Elsevier Ltd.
- [2] Rajvinder Singh, Suraj Pal, Arashdeep Singh. "Analysis of IC Engine Air Cooling of Varying Geometry and Material" Imperial Journal of Interdisciplinary Research (IJIR) Vol-2, Issue-12, 2016 ISSN: 2454-1362, http://www.onlinejournal.in.
- [3] Hrushikesh S Lokhande, Pramod K Jadhao. "Thermal Analysis of Engine Cylinder Head by Perforations" international journal of advanced research in science and engineering, volume no-7 special issue no. (03), January 2018.
- [4] Kota LeelaSaiBharath, V Pradeep Kumar. Design and Analysis of a Rectangular Fin with Comparing by Varying Its Geometry and Material, With Perforation and Extension" Turkish Journal of Computer and Mathematics Education, Vol.12 No.2 (2021), 3039 – 3050, Accepted: 27 February 2021; Published online: 5 April 2021.
- [5] T R Chinnusamy, E Balaji, V Vignesh, T Karthikeyan, M Krishnan. ".Design Analysis of Cooling Fins for Two-Wheeler using ANSYS", Vol. 2(1) Jan – Jun 2019: ISSN (Online): 2581-7019 @ Guru Nanak Publications, India.
- [6] Mr. Atul Chandanshive, Prof. Dr. S. M. Pise. "Thermal Analysis and Optimization of Two-Wheeler Engine Cylinder Fins", International Research Journal of Engineering and Technology(IRJET), Volume: 06 Issue: 05 | May 2019, e-ISSN: 2395-0056, p-ISSN: 2395-0072
- [7] Pulkit Sagara , Puneet Teotiab , Akash Deep Sahlotc , H.C Thakurd. "Heat transfer analysis and optimization of engine fins of varying geometry" , materialstoday proceedings , Volume 4, Issue 8, 2017, Pages 8558-8564.
- [8] Chandrakant R Sonawane, Pratyush Rath, Nishant Vats, Shreyas Patekar, Prakhar Verma, Anand Pandey. "Numerical simulation to evaluate the thermal performance of engine cylinder Fins: Effect of fin geometry and fin material", 2214-7853/ 2021 Elsevier Ltd. All rights reserved.
- [9] Vinoth kanna. "Modelling and Analysis of the thermal behaviour of air-cooling system with fin pitch in IC Engines", ISSN: 0143-0750 (Print) 2162-8246 (Online) Journalhomepage: http://www.tandfonline.com/loi/taen20.
- [10] L.Natrayan,G. Selvaraj,N. Alagirisamy, M.S.Santhosh. "Thermal Analysis of Engine Fins with Different Geometries", International Journal of Innovative Research in Science, Engineering and Technology, (An ISO 3297: 2007 Certified Organization), Vol. 5, Issue 5, May 2016.